A Smooth Position-Force Controller for Asbestos Removal Manipulator

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Abstract—In robotic grinding tasks, the robot controller must reactively adapt to sudden changes in the environment and be able to handle uncertainties in texture, change in materials, and disturbances caused by vibrations, impacts and friction of the grinding tool operating at high rotational speed. This requires managing sudden signal changes in sensor data. In this paper, we present a smooth hybrid force/position controller based on distance measurements from radars, enabling the robot to achieve a stable interaction with the environment while grinding an unknown three-dimensional surface. The control uses an actively compliant wrist that maintains a desired force centered on the disc and normal to the surface. Our controller is based on a smooth transition between free space and contact modes, significantly reducing the impact force. Additionally, the vibro-dynamic effects are suppressed and smooth environmental tracking is ensured by the impedance/admittance control of the wrist. This framework is validated on a 6-dof anthropomorphic arm through dynamic simulation. The controller is able to adapt reactively to abrupt disturbances in the environment (ex: sudden impacts on the disc) while ensuring good position and force tracking performance.

I. INTRODUCTION

Despite the advents in industrial automation, robotic solutions are not yet commonly used in construction and demolition industries. However, implementation of robots in civil engineering is strongly growing, as the International Federation of Robotics (IFR) survey showed about 568 construction and demolition systems were supplied in 2015, and the number is estimated to increase to 2,800 in 2019 [1]. Currently, most tasks are performed manually with the help of conventional electro-mechanic and hydraulic tools. For example, asbestos removal by grinding is still performed by human operators, subjecting them to health hazards like cancer, since the asbestos fibers can infiltrate into respiratory system even with the use of protection. Additionally, the cleaning productivity is limited by human performance while the surface area of contaminated flats is considerable. Thus, the automation of this task to achieve efficient asbestos removal from a real world rehabilitation site, without endangering the human life is recommended [2], [3]. A conceptual design of the system can be represented as shown in Figure 1. In order to develop such system, investigation is carried out to build a multi-modal control based on the association between radar and force informations that ensure a constant grinding force of desired magnitude and orientation. For this grinding task, the challenge is to achieve a smooth cleaning of the surface while
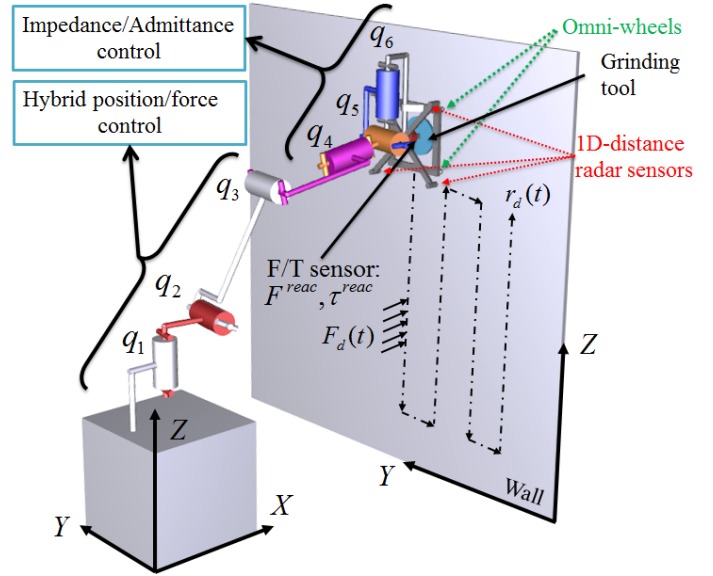

Fig. 1: Kinematic structure of the robotic system equipped with a grinding tool, several 1D-distance radar sensors around it for attitude measurement, and a force/torque sensor to perform asbestos removal on the wall

adapting to it and to the random impacts from the large grinding disc where the latter can not be neglected. The issues mentioned direct the problem into two main tracks: control strategies for robot-environment interaction and robotic grinding. Concerning the interaction with the environment, many important works have been carried out in the literature. The control strategies proposed can be grouped into two categories: indirect and direct force control, in which the former achieves motion control without explicit closure of a force feedback loop while the latter closes the force loop on the measured force value [4]. To the first category belong impedance and its alternatives, explained by Hogan in [5], where he introduced a method for a manipulator to interact dynamically with its environment. A meaningful set of controllers and their respective design specifications to assure compliant motion with stability robustness were described and analyzed in [6], [7] and a formalism for specifying compliant motion tasks and tracking was explained in [8]. The capabilities of impedance control have been enhanced by providing means of force

tracking to keep a desired contact force with the environment despite lacking the knowledge of its stiffness and position [9]. This framework has been proposed by two simple on-line schemes. The first is based on generating the reference position on-line as a function of the force-tracking error and the second estimates environmental parameters using an indirect adaptive approach. Moreover, a force limited impedance and position limited force control were proposed in [10] by implementing an impedance controller as an inner loop and integral force controller as an outer loop. This allowed changing the behavior based on the limited external force. Hence, a good position, impedance and force tracking is obtained. A more advanced force tracking impedance control based on adaptive methods has been proposed in [11]. It worked well for abrupt changes in the environmental stiffness. Then the author of [12] proposed a predictive force control that used a fuzzy scaling machine. The controller performance seems good but it needs the estimation of the environmental stiffness. In [13], [14] the controller is based on learning to reduce the error and energy and make the robot behave like a human while interacting with unknown environments. It showed that the robot is resistant to uncertainties after learning but off-line training is essential. Carrying on performance improvement, [15] includes the redundancy resolution in the controller and solves the interaction by allowing the end-effector to comply according to an impedance control law defined in the Cartesian space. More works have been done on performance issues of the impedance and regarding external forces as presented in [16]–[20] where the researchers combined impedance with extra vision and tactile sensors.

On the other hand, hybrid position/force falls within the direct force control category. When the manipulator is in contact with the environment, the end-effector coordinate space can be decomposed into position and force subspaces, then control in each subspace can be done [21], [22]. The explicit model based hybrid control of rigid robot in contact with rigid environments has been studied in [21], [23], [24] and with compliant environments [25]–[27]. A review about the approaches used for controlling robots in constrained motions can be found in [28]. These hybrid controllers suffer from stability issues because of switching, impacts and uncertainties, which challenged researchers to investigate solutions. Attempts have been made to solve these issues using active stiffness and impedance control [29], [30]. Other solutions reduced environmental stiffness using mechanical means such as soft sensors or compliant covers [31], [32]. Passive and dynamic damping control was used [33]–[35] as an alternative.

As many studies deduced, using only one sensor limits the tasks the robot can do. Hence to ameliorate the force and position performances in unknown environments, the hybrid control is extended with vision [36]–[38]. The controllers suffered from disturbance and the closed loop stability was not proven. Then the uncertainties of the robot and the environment were taken into account and more work on the precision was done in [39]–[46]. Moreover, the contact problem has been addressed in the literature [37], [47], [48]. In [40], the force was added after contact to reduce impact, while in [42], the control switches from "position" before contact to "force" control after-wise. The author in [49] used motion control while approaching, followed by impact loading control that dissipates the impact force by setting the force command value negatively proportional to the velocity of the end-effector upon contact. In [50], the contact velocity is decreased based on vision and a rubber damper is added to reduce impact.

Vision is very useful for locating the environment, and determining the relative position and orientation of the robot's end-effector. It has been widely used to improve the human-robot and robot-environment interactions. [51] shows how vision can be used to detect the system states to ensure safe interaction with the human. In [19], [45], [52], vision sensors were exploited in applications that require interaction with the environment. Although vision is powerful in canceling unwanted control behaviors and in adding the knowledge about environment, it is usually not sufficient when the environment is rugged and dusty. Hence, in Section II of this paper we present a unified control strategy based on 1-D radars, furnished with smooth transition function to minimize the impact force as it is explained in Subsection II-C. The grinding forces similar to the ones analyzed in [53] are limited by the controller. A detailed model is presented in Subsection II-F for grinding walls with abrasive disc, where it is used to validate the controller. Hence, taking the advantage of force measurements and radar information, better fitting to the surface can be achieved while grinding and ensuring a normal contact force on the wall coinciding with the tool center.

II. CONTROL FRAMEWORK

Without loosing generality, the control scheme presented in this paper can be applied to any robotic arm where position and orientation can be decoupled. The first three joints of the arm are used to ensure a desired force on the wall and trajectory tracking using hybrid position-force control. The control of wrist joints is based on admittance to ensure the adaptation of the tool to the wall as shown in Figure 1. The control scheme is explained in the following subsections:

A. Hybrid Control in Operational Space

In complex and unpredictable environments, e.g grinding operation, the end-effector motion may be subjected to sudden disturbances occurring due to excessive forces. Thus, in order to accommodate these unexpected interactions, the end-effector motion must be adapted by on-line modifications. Expressing the task in operational space requires a precise control of the end-effector motion, which can be achieved by the hybrid control proposed in [4]. Based on dynamics, the controller can be expressed as follows:

$$h_c = \Lambda(q)S_v\alpha_v + S_f F_{cmd} + \mu(q,\dot{q}) \qquad (1)$$

where $h_c \in R^n$ (applied on the 1st 3 joints and $n = 3$) denotes the output wrench of the end-effector in operational space; q corresponds to joint values, S_v and S_f are the selection matrices of position and force controlled directions respectively; α_v
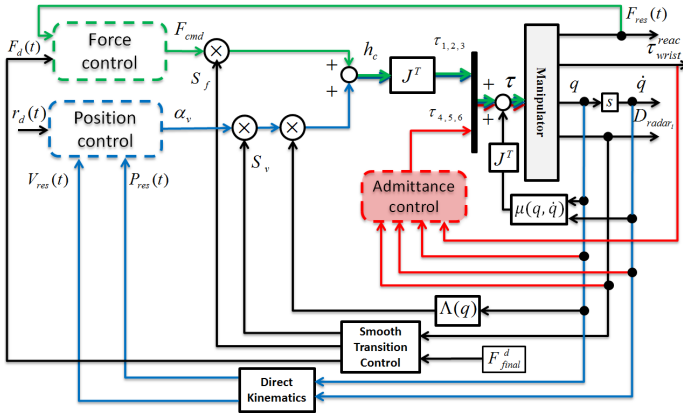
Fig. 2: Hybrid controller block diagram: force control loop (green), position control loop (blue), admittance control loop (red) (refer to Section II)

and F_{cmd} are the acceleration and force commands respectively, $\Lambda(q)$ is the pseudo-inertia matrix defined by:

$$\Lambda(q) = JH(q)^{-1}J \qquad (2)$$

with J denoting the $(n \times n)$ kinematic Jacobian matrix, H is the $(n \times n)$ robot inertia matrix; μ is the $(n \times 1)$ function to compensate for Coriolis, gravitational and friction forces in the workspace. It is defined by:

$$\mu(q, \dot{q}) = \Gamma(q, \dot{q})\dot{q} + \eta(q) \qquad (3)$$

where Γ is the wrench mapping the centrifugal, Coriolis and friction effects $c(q, \dot{q})$ from joint space into operational space:

$$\Gamma(q, \dot{q}) = J^{-T}c(q, \dot{q})J^{-1} - \Lambda(q)\dot{J}J^{-1} \qquad (4)$$

and η is the wrench mapping the gravitational effects $g(q)$ from joint space into the operational space as:

$$\eta(q) = J^{-T}g(q) \qquad (5)$$

Finally the joint torques τ can be calculated by:

$$\tau = J^T h_c \qquad (6)$$

The control loop expressed in equation (1) allows full decoupling between the force and velocity controlled subspaces (refer to Figure 2).

B. Force and Position Control Loops:

The desired force F_d can be achieved by setting:

$$F_{cmd} = F_d(t) + K_{PF}[F_d(t) - F_{res}(t)] \qquad (7)$$

F_{cmd} is the command to force controller, F_{res} is the reaction force value and K_{PF} is a suitable positive-definite gain matrix. The proportional feedback is able to reduce the force error due to disturbance forces.

Position control can be achieved by setting:

$$\alpha_v = \ddot{r}_d(t) + K_{Dr}[\dot{r}_d(t) - V_{res}(t)] + K_{Pr}[r_d(t) - P_{res}(t)] \qquad (8)$$

V_{res} and P_{res} are the velocity and position response of the end-effector computed by the direct kinematics; K_{Dr} and K_{Pr} are

suitable gain matrices; $\ddot{r}_d(t)$, $\dot{r}_d(t)$ and $r_d(t)$ are the desired acceleration, velocity and position tracking inputs, obtained from the trapezoidal trajectory generator with continuous acceleration as detailed in [54].

C. Smooth Transition Control

As described in Section I, the switching problem is of main concern. In order to avoid the discontinuous switching between the controllers, and to reduce the impact force, a new strategy is introduced to change the selection matrix element $S(i, j)$ corresponding to the desired direction of motion from 0 to 1 smoothly. This way, the controller inputs are continuous and the control scheme smoothly flips from full position to hybrid control according to the distance from the grinding tool to the wall D_{wall} obtained from radar readings:

$$D_{wall} = min(D_{radar_1}, D_{radar_2}, D_{radar_3}, D_{radar_4}) \qquad (9)$$

Hence, exponential variation has been chosen for this issue as:

$$S_f(i, j) = S_{f_0}e^{k_a * D_{wall}} \qquad (10)$$

$$S_v(i, j) = 1 - S_f(i, j) \qquad (11)$$

with

$$k_a = -\frac{log(\frac{S_{f_{final}}}{S_{f_0}})}{D_{final}} + \varepsilon \qquad (12)$$

$$\varepsilon = D_{wall}^{final} - D_{wall}^{initial} \qquad (13)$$

$S_{f_{final}}$ is chosen as a small scalar close to 0, $S_{f_0} = 1$ and the impact control is regulated by ε according to the distance range defined by $[D_{wall}^{final} - D_{wall}^{initial}]$. When $S_f(i, j)$ reaches 1, F_d starts from 0 to reach the maximum desired value:

$$F_d(t) = \begin{cases} 0 & \text{if } t \leq t_{impact} \\ r_f(t - t_{impact}) + F_0 & \text{if } t_{impact} < t \leq t_{impact} + w \\ F_{final}^d & \text{if } t > t_{impact} + w \end{cases} \qquad (14)$$

where $r_f = \frac{F_{final}^d}{w}$ is the force rate, F_0 is the initial value of F^d and w is the desired period to reach the maximum force.

D. Impedance/Admittance Based Orientation Control

In order to ensure soft interaction with the environment and stability against impacts, joint based impedance/admittance controller has been utilized for the wrist joints [55]. It ensures the tool-wall adaption with minimal torques (refer to Figure 3). The torque acting on the end-effector when moving along the surface is:

$$\tau_{ext} = K_{reac}\tau_{wrist}^{reac} \qquad (15)$$

K_{reac} is gain of reaction forces, τ_{wrist}^{reac} is the reaction torque on the wrist, obtained from the torque sensor. Hence, the controller can be realized based on accelerations references as:

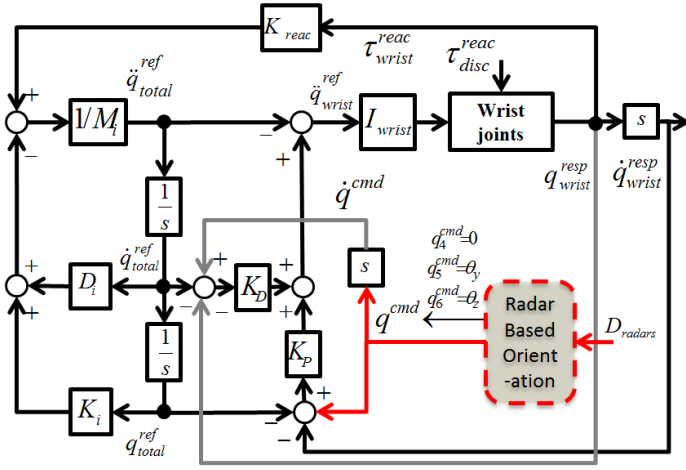$$\ddot{q}_{ref}^{total} = \frac{1}{M_i}(\tau_{ext} - D_i\dot{q}_{ref}^{total} - K_iq_{ref}^{total}) \qquad (16)$$

Fig. 3: Joint based admittance controller block diagram (refer to Sections II-D, II-E)

where M_i, D_i and K_i are the impedance gains. They can be tuned by fixing the 2nd order dynamics as:

$$\frac{\ddot{q}_{wrist}^{ref}}{\tau_{ext}} = \frac{1}{M_i s^2 + D_i + K_i} \tag{17}$$

The natural frequency ω_i and the damping ratio ζ_i are defined as:

$$\omega_i = \sqrt{\frac{K_i}{M_i}} \text{ and } \zeta_i = \frac{D_i}{2\sqrt{M_i K_i}} \tag{18}$$

Finally, the motors angular acceleration can be defined as:

$$\begin{aligned} \ddot{q}_{wrist}^{ref} = & -\ddot{q}_{ref}^{total} + K_p(q^{cmd} - q_{ref}^{total} - q_{resp}^{wrist}) \\ & + K_d(\dot{q}^{cmd} - \dot{q}_{ref}^{total} - \dot{q}_{resp}^{wrist}) \end{aligned} \tag{19}$$

and the torque for wrist joint as,

$$\tau_{wrist}^{ref} = I_{wrist}\ddot{q}_{wrist}^{ref} \tag{20}$$

where K_p and K_d are the PD gains respectively, and I_{wrist} is the wrist inertia matrix.

E. Radar Based Orientation

Based on radar measurements, the absolute rotations between the tool and the wall, shown in Figure 4 can be calculated as:

$$\begin{cases} \theta_y & = sign(r_3 - r_2)(\frac{\pi}{2} - \gamma) \\ \theta_z & = sign(r_1 - r_2)(\frac{\pi}{2} - \alpha) \end{cases} \tag{21}$$

With

$$\gamma = \begin{cases} tan^{-1}(\frac{d_{r_2 r_3}}{r_2 - r_3}) & \text{if } r_3 \leq r_2 \\ tan^{-1}(\frac{d_{r_2 r_3}}{r_3 - r_2}) & \text{if } r_3 > r_2 \end{cases}$$

$$\alpha = \begin{cases} tan^{-1}(\frac{d_{r_1 r_2}}{r_2 - r_1}) & \text{if } r_1 \leq r_2 \\ tan^{-1}(\frac{d_{r_1 r_2}}{r_1 - r_2}) & \text{if } r_1 > r_2 \end{cases}$$

r_1, r_2 and r_3 are the radar measurements respectively, and $d_{r_i r_j}$ is the distance between radars i and j (refer to Figure 4).
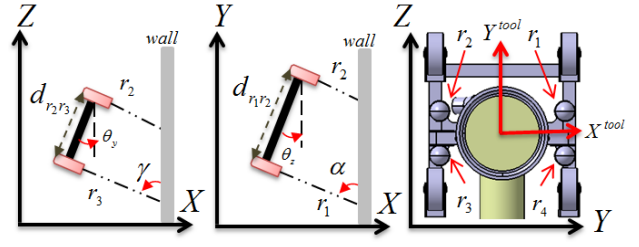

Fig. 4: Tool absolute orientation

TABLE I: Parameters used in grinding model.

Variable	Unit	Explanation
MRR	m^3/s	Material removal rate
U	J	Kinetic energy
ω_0	rad/s	Free rotational speed of the disc
F	N	Applied force
m	Kg	Mass
t	m	Thickness of the cut ahead
R_1, R_2, R_3	m	Contact radii on the disc
d	m	Depth of the cut
b	m	Width of the cut
D	m	Disc diameter
υ	m/s	End-effector velocity
a	m/s^2	End-effector acceleration
$A_{contact}$	m^2	Wall-disc contact area
K_1	$J.m$	Constant relating the cut ahead to the energy
β	$rad/(m.N.s)$	Speed reduction to torque constant
μ		Friction constant
δ	N.m	Delta function
n	N.m	Limited noise

F. Disturbance Model of Grinding Torque

In order to test the control response, and to simulate it against noises and impacts of the grinding process, a torque disturbance model is needed. In [53], a metal grinding process was modeled. The normal force on the disc was estimated based on it's physical parameters. Moreover, speed reduction value is formulated based on this normal force as:

$$speed\ reduction\ value = \beta R \mu F_{norm} \tag{22}$$

R is the radius to the point of contact, the variables β, μ and F_{norm} are defined in in Table I. In [56], the grinding force was estimated as a function of tool kinetic energy (U), material removal rate (MRR) and the disc speed on the wall (υ_{norm}) as:

$$F_{norm} = U * \frac{MRR}{\upsilon_{norm}} \tag{23}$$

In our case, as the disc lies in plane of the wall, force exerted on the disc and it's speed are known. The interest here is to study the effect of grinding parameters defined in Table I and Figure 5 on the generation of the reaction torque. A model has been developed based on existing studies as:

$$\tau_{disc}^{reac} = sign(F_{norm}) * U * \left(\frac{(MRR)_{tan} + (MRR)_{norm}}{\omega_0 - \omega_{overload}} \right) + N(n, \delta) \tag{24}$$
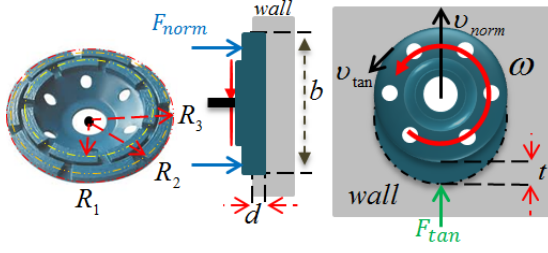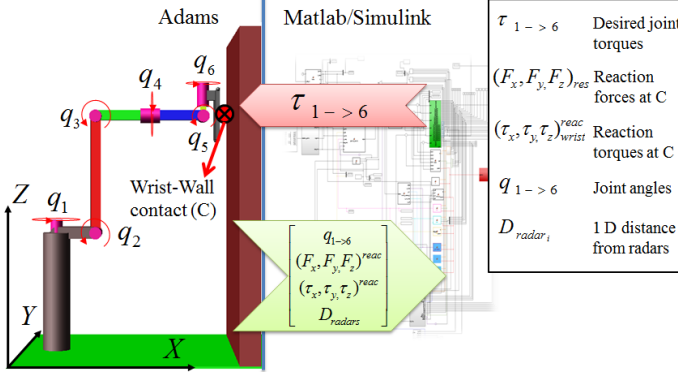
Fig. 5: Grinding parameters

Fig. 7: Position command-response

Fig. 6: Adams/Matlab co-simulation

Fig. 8: Position command-response errors

Fig. 9: Velocity command-response

with,

$$
\begin{cases}
\omega_{overload} &= \beta\mu(R_1 F_{norm}^{R_1} + R_2 F_{norm}^{R_2} + R_3 m a_{tan}) \\
(MRR)_{tan} &= v_{tan} d b \\
(MRR)_{norm} &= v_{norm} D A_{contact} \\
U &= K_1/t
\end{cases}
$$

Where *tan* and *norm* subscripts stand for tangential and normal directions respectively. Additionally, random impacts δ and limited noise n have been included in the noise function N to represent a real case scenario.

III. SIMULATIONS

The control framework in Section II is implemented for a 6-R robot using Adams-Matlab co-simulation (Figure 6). The system is simulated with a trapezoidal trajectory generator that provides continuous acceleration in the variable velocity phases and constant speed otherwise. The robot is commanded to apply a force of 90 N on a desired path on the wall. It starts from free space and goes into the wall by a smooth transition from position to force control avoiding impact and maintaining the desired force. The admittance controller ensures centering the force and adapting to the wall.

A. Hybrid Force-Position Control Behavior

The command-response of trajectory position and velocity are shown in Figures 7 and 9 respectively. The controller shows good performance in free space and after contact, limiting the position errors to 0 mm before and 5 mm after contact at $t = 1.4$ s as shown in Figure 8. This variation in position errors is due to the 0.4 N dynamic friction force between
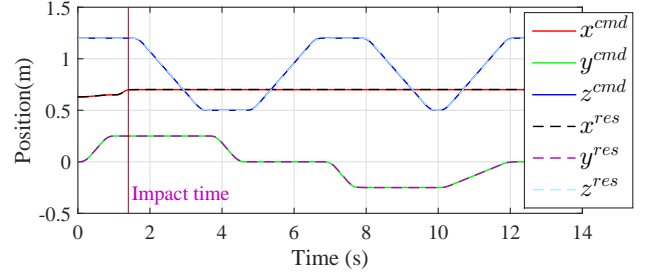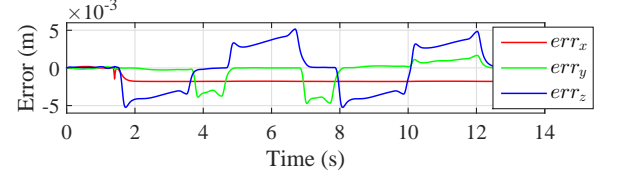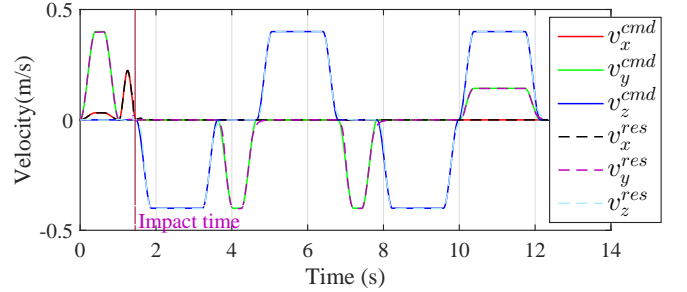
the tool and wall, that is defined in Adams. Consequently the velocity errors are between 0.035 m/s and −0.05 m/s after impact and are shown in Figure 10.

Concerning the force control performance, the force command versus response are plotted in Figure 11. The control with smooth transition in Figure 11b shows negligible impact force when touching the wall. Then the desired force reference value is reached in a behavior similar to a step function. The force value is maintained along the path with an error less than 2 N as shown in Figure 12b. The smooth transition control shows better values compared to the ones obtained by direct switching in Figure 11a, where the impact is clearly high and the force error can exceed 100 N as shown in 12a. Thanks to the smooth transition control presented in II-C that flips smoothly from position into force control in a unified manner that avoids switching as Figure 13 shows.

B. Admittance Control Behavior

As described in II-D, the wrist motors are based on acceleration control (eq.19) in addition to the torque sensor to measure the reaction torques τ_{wrist}^{reac}. The controller gains M_i, D_i and K_i are tuned for 2nd order differential equation to ensure stability and equivalent distribution of contact forces on the
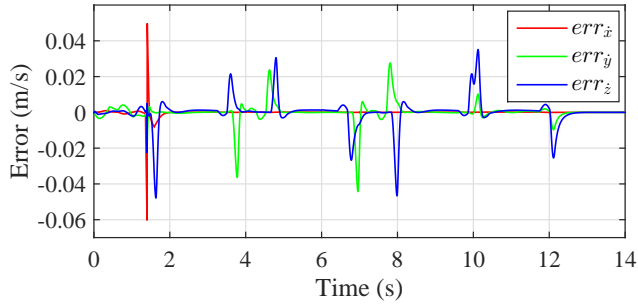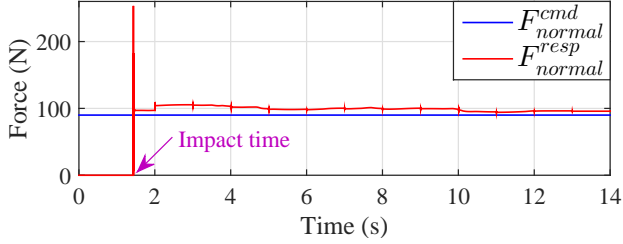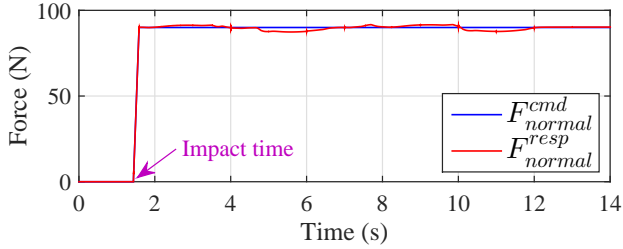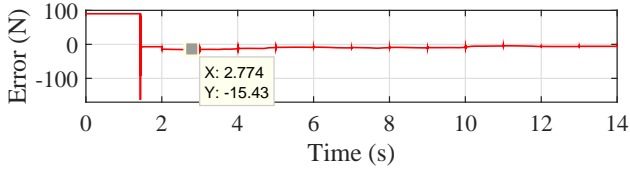
Fig. 10: Velocity command-response errors

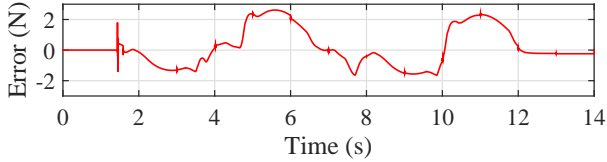(a) Force command-response without smooth transition

(b) Force command-response with smooth transition

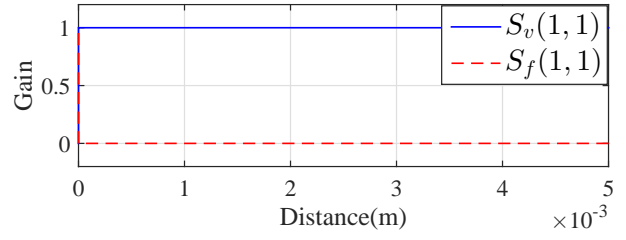Fig. 11: The command-response of normal force on the wall

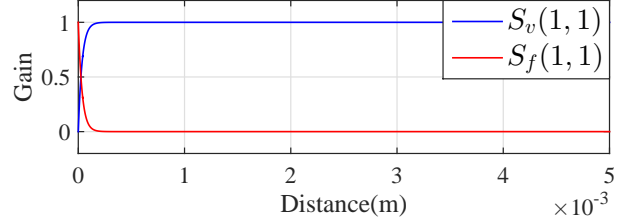(a) Force command-response error without smooth transition

(b) Force command-response error with smooth transition

Fig. 12: Normal force command-response errors

end-effector (eq.17 & 18). K_p and K_d are proportional and derivative gains tuned for good tracking of the reference values generated from commands and external torques (eq.16 & 19) as shown in Figure 17, where the joint positions are modified based on external torques. The effectiveness of the admittance

(a) Gains for direct switching from position to force control

(b) Gains for smooth transition from position to force control

Fig. 13: Smooth switching from position to force control. (1,1) is the index to the first element of the selection matrix
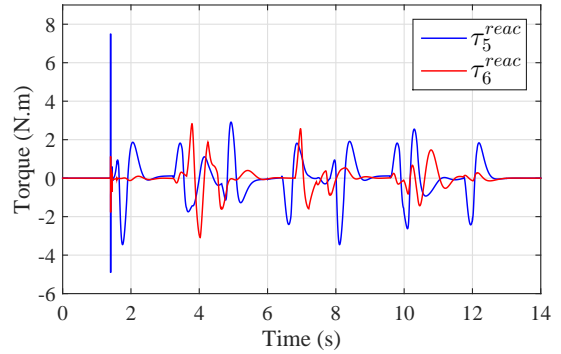

Fig. 14: The reaction torques on 5th and 6th joints

controller in adapting the tool to the wall, can be deduced from the fact of keeping minimal torques on 5th and 6th joints, that are responsible for the pitch and yaw of the tool as Figure 14 shows. Figure 15 shows how the controller maintains the force centered as the zero-moment-point of the tool-wall contact gets about the center of the tool (except for the first contact) overcoming lateral forces.

Adding external torque to the disc with some impacts, mainly results on the 4th joint because of the mechanical linkage of the wrist that is similar to universal joint that couples rotation between two drive-line shafts: the base shaft (axle of the disc) and the follower shaft (joint 4). The results are shown in Figure 16 where the τ_4 in 16a shows the behavior of joint 4 without external torque. However, when the end effector is subjected to external torque as in 16b, τ_4 acts as shock absorber and it can suppress noise and impacts of the disc as 16c shows, thus avoiding high impacts on the joint.
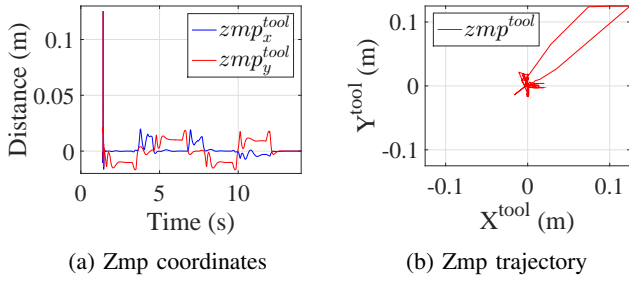
(a) Zmp coordinates

(b) Zmp trajectory

Fig. 15: Zero-moment-point on the end-effector expressed in the tool frame shown in Figure 4

(a) Response of τ_4 without the disc reaction torque

(b) Disc reaction torque plus noise and random impacts

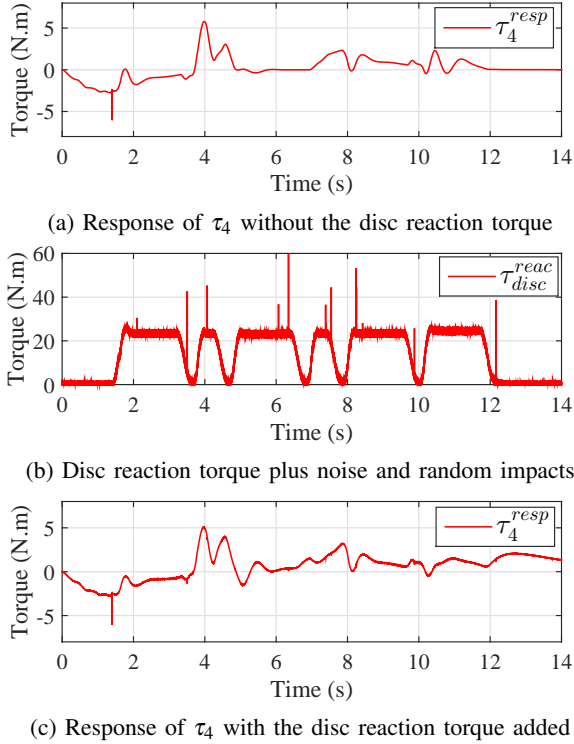(c) Response of τ_4 with the disc reaction torque added

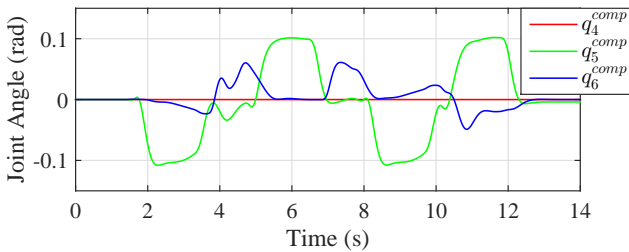Fig. 16: Suppressing the disc noise and random impacts by impedance control

Fig. 17: Position compensation by admittance control

IV. CONCLUSIONS AND FUTURE WORK

This article presents a smooth position-force hybrid controller that can reactively adapt to sudden changes in the environment and the resulting impacts while grinding. The controller is based on 1D-distance measurement radars and active admittance for tracking unknown three-dimensional surfaces while maintaining a desired force centered on the disc and normal to the surface. The switching problem is overcome by proposing a smooth transition control. The controller changes smoothly between free space and contact modes, thus reducing impact force close to zero and avoiding uncertainties. Additionally, a disturbance model to estimate the reaction torque of the tool has been developed in order to test the controller capable of suppressing vibro-dynamic effects. The control is validated by simulations on a 6-dof anthropomorphic arm. The results show good position and force tracking performances and impact force close to zero. In the near future, the controller will be implemented and tested by real robot on flat and curved surfaces in order to better evaluate and ameliorate its performance.

ACKNOWLEDGMENT

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